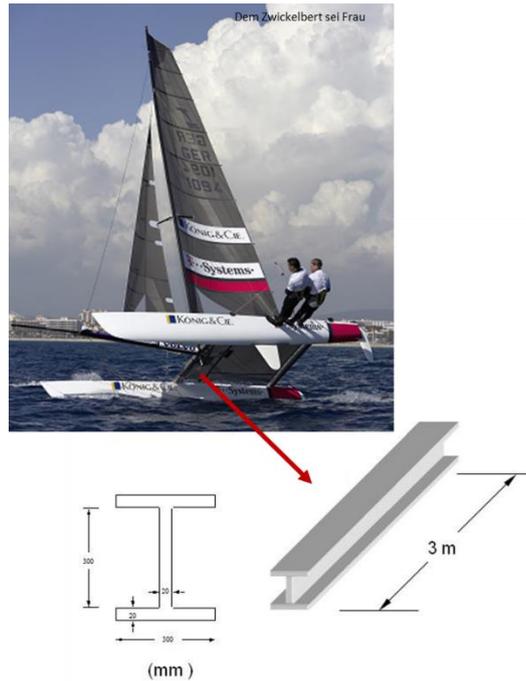


## CHAPTER 4

### AUTO EVALUATION EXERCISE

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A catamaran is being designed using composite materials. The union between the two hulls is done via a 3 meters long beam with a double T section. The beam is manufactured by a symmetrical laminate composed by the same number of plies at  $0^\circ$ ,  $45^\circ$  and  $-45^\circ$ .



Modelling the structure as a single-span beam supported at its ends and subjected to a uniformly distributed load:

1. Determine the maximum stresses and the beam sections in which they appear.
2. Assuming that the bending moment would be supported solely by the flanges, propose a model to express this moment as a pair of axial forces acting on each flange.
3. Calculate the strain distribution in the upper flange.
4. Estimate the stresses that appear on the plies at  $0^\circ$  of the laminate.
5. Considering exclusively the plies at  $0^\circ$ , determine the load that produces the failure.

DATA:

Material: Kevlar/epoxy

$$E_1 = 85 \text{ MPa}$$

Density:  $1350 \text{ kg/m}^3$

$$E_2 = 5,6 \text{ GPa}$$

$X = 1410 \text{ MPa}$

$$\nu_{21} = 0,34$$

$X' = 280 \text{ MPa}$

$$G_{12} = 2,1 \text{ GPa}$$

$Y = 28 \text{ MPa}$

$$Y' = 141 \text{ MPa}$$

$S = 45 \text{ MPa}$

Stiffness matrix of the lamina

$$[Q] = \begin{bmatrix} 85,65 & 1,92 & 0 \\ 1,92 & 5,64 & 0 \\ 0 & 0 & 2,10 \end{bmatrix} \text{ GPa}$$

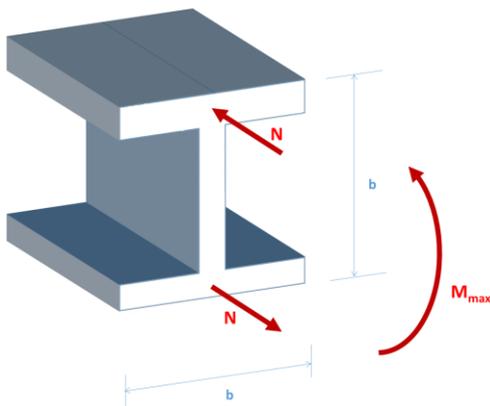
Ans:

- Using the Equilibrium equations:

Maximum shear force (In the supports):  $\frac{qL}{2}$  (in N)

Maximum bending moment (in the middle cross-section):  $\frac{qL^2}{8}$  (in N/m)

- Assuming that the bending moment would be supported solely by the flanges, the bending moment can be represented by a pair of forces.



$$N = \frac{M}{b}$$

According to laminate theory

$$N_x = -\frac{qL^2}{8b^2} \text{ (top flange)}$$

$$N_x = \frac{qL^2}{8b^2} \text{ (bottom flange)}$$

### 3. Laminate [0/+45/-45]<sub>ns</sub>

Plane stiffness matrix:

$$[A] = 2 \cdot ([\bar{Q}]_0 + [\bar{Q}]_{45} + [\bar{Q}]_{-45}) \cdot n \cdot h$$

$$[A] = \frac{e}{3} \begin{bmatrix} 137.416 & 45.280 & 0 \\ 45.280 & 57.406 & 0 \\ 0 & 0 & 45.828 \end{bmatrix} \text{ (GN/m)}$$

$$e = 6nh$$

Strain in the flanges:

$$\{\varepsilon\} = \begin{Bmatrix} 1.475 \cdot 10^{-9} \\ -1.163 \cdot 10^{-9} \\ 0 \end{Bmatrix} N_x$$

### 4. Stress in plies at 0°

$$\{\sigma\} = [\bar{Q}]_0 \cdot \{\varepsilon\}$$

$$\{\sigma\} = \begin{Bmatrix} 124.1 \\ -3.79 \\ 0 \end{Bmatrix} N_x \text{ (Pa)}$$

### 5. Applying the maximum stress criterion:

$$q_{\max} = 180 \text{ kN/m}$$